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STATIC AND DYNAMIC BUCKLING OF SHALLOW SPHERI-CAL SHELLS SUBJECTED TO AXISYMMETRIC AND NEAR-LY AXISYMMETRIC STEP PRESSURE LOADS USING SA-TANS-IIA, A MODIFIED VERSION OF SATANS-II

Michael D. Shutt

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THESIS

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by

Michael D. Shutt

December 1976

Thesis Advisor:

Robert E. Ball

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by

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Submitted in partial fulfillment of the requirements for the degree of

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ABSTRACT

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LIST OF SYMBOLS

b	nondimensional inplane stiffness
E	the modulus of elasticity of the shell
Н	the rise of the spherical cap at the pole
h	the thickness of the shell
m	the mass density of the shell
M s	the meridional bending moment per unit length
n	the Fourier index
P	a nondimensional applied load
PCRIT	the nondimensional critical pressure
oP	the classical buckling pressure of a complete
	sphere
(n)	a column matrix containing the coefficients
	th of the n term in the series expansion of the
	applied load
r	the normal distance from the axis of revolution
	to the surface of the cap
r	the normal distance from the axis to the cap
	in the base plane; the maximum value of r
R _s , R _e	the radii of curvature in the s and 0
	directions, respectively
s	the meridional distance along the surface
	of the shell
t	the nondimensional time
T	the time
T	a reference time



- U, V, W = the displacements in the s, e and 3 directions, respectively
- u, v, w = nondimensional series coefficients of U, V, W
- \overline{V} = the peak in the time history of the parameter \overline{V}
- w = the displacement in the 3 direction in the n harmonic
- ε (n) = the nondimensional parameter governing the
- = the nondimensional parameter governing the
 magnitude of the load applied in the asymmetric
 harmonics
- f = the coordinate normal to the surface of the shell
- = the circumferential angle measured about the axis of revolution
- λ = a nondimensional geometric parameter used
 to describe the spherical cap
- y = Pcisson's ratio



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I. INTRODUCTION

In 1973 a digital computer study was presented by Ball and Burt [1] for the dynamic buckling load of clamped shallow spherical shells subjected to axisymmetric and nearly axisymmetric step-pressure loads. A static buckling analysis of the same spherical shells had been carried 1970 by Stilwell and Ball [2]. In these two studies the digital computer program SATANS-I [3] was used to calculate the critical buckling pressures for a large range of shell Other studies of the buckling of shallow shells have been conducted by Huang [4,5], by Stephens and Fulton [6], by Lock et al. [7], by Stricklin [8], and most recently by Akkas [9]. In Reference 1 the results from these other studies, except for those by Akkas, are compared with results from SATANS-I for both static and dynamic buckling. In the axisymmetric static analysis the comparison with the obtained by Huang [4] revealed that the SATANS-I results results were higher than Huang's results for several the dynamic, axisymmetric buckling analysis the SATANS-I results again either agreed closely with, somewhat higher than , the results by Huang [5], Stephens and Fulton [6], and Stricklin [8]. However, it was that there was a general lack of consistent agreement among any of the sets of results. As a consequence, appeared at that time that the axisymmetric buckling problem had not yet been totally resolved and that additional studies would be appropriate.

In the asymmetric dynamic buckling analysis of Reference 1 the few comparisons that could be made for the critical load also indicated that the SATANS-I results may be too



high. A comparison of the recent estimates for the asymmetric dynamic buckling load obtained by Akkas [9] with the SATANS-I results also reveals the SATANS-I results to be well above those of Akkas [9]. However, it should be noted that the results obtained by Akkas were from his attempt to obtain a lower bound on the critical asymmetric load. bound on the buckling load is obtained without the execution of a complete transient response analysis on the asymmetric part of the response of the shell, as is done in SATANS-I. In Akkas' analysis (Problem 1) the transient nonlinear axisymmetric response is computed, and a determinant is examined for possible bifurcation into asymmetric moticn at each time step. The minimum load at which the determinant becomes zero is defined as the lower bound of the critical load.

As a consequence of the generally high buckling loads predicted by SATANS-I, a re-examination of the static and dynamic buckling of the shallow spherical shell was made in an attempt to determine the possible cause, or causes, of the high buckling loads. In our search we discovered that a modification of the manner in which the pole conditions are numerically approximated significantly lowered the buckling loads to values that are now in good agreement with the other results. The new procedure for handling the pole condition is given in section III of this thesis. The new buckling results are given in section V.

In addition to the pole condition modifications and the new buckling results the author has also made another significant change to the SATANS family of codes. In particular, the SATANS-II program for the geometrically nonlinear analysis of totally arbitrarily loaded shells of revolution, developed by Ryan [10] in 1972 to handle more complex and larger problems, was modified to make the computer memory requirement a variable quantity. This



quantity is specified by the user to fit the particular problem being run. It eliminates the large core requirement of SATANS-II for small problems and allows for much larger problems to be solved than could be solved by SATANS-II. The new program with the pole condition and memory modifications will hereafter be called SATANS-IIA. It is described in section II.



II. DESCRIPTION OF SATANS-IIA

SATANS-II was developed by Ryan [10] from SATANS-I incorporated the full trigonometric expansion of the applied load and solution vector, and introduced the handling of imperfections into the code. These modifications allow the analysis of shells under totally arbitrary loads, as well as imperfection actual shells studies on with measured imperfections [11]. Unfortunately, the original cards for SATANS-II was destroyed. Professor Johann Arbocz of CALTECH had a listing of SATANS-II and punched a deck cards with the changes to SATANS-I given in that listing. A copy of this deck was sent to Professor Ball. These cards have been added by the author to the original SATANS-I described by Ryan [10] and a complete version of SATANS-II has been reconstructed. SATANS-IIA is a modification by the author of the reconstructed SATANS-II program. A listing of SATANS-IIA can be found in Appendix A. The listing contains an example problem for the dynamic analysis of a truncated cone subjected to an impulsive loading which is along the meridian and varies in a cosine circumference. distribution over one-half of the by the Lockeed problem is a sample problem suggested Missiles and Space Corp. [12]. A condensed version of the output from the example problem is given in Appendix Input data preparation for SATANS-IIA can The which includes Appendix C. basic users manual. subroutines and the theory of the preparation of input program, is contained in Reference 3, which can be obtained (M70-10098, LAR-10736), or ASIAC [13]. A through COSMIC users manual which includes preparation and handling imperfection data within the SATANS programs can be found in



Ref. [10]. The above information, along with the following discussion, will inform the user on the capabilities and proper use of SATANS-IIA.

The modification of the SATANS-II program to make its core requirement variable was accomplished by putting in a single dimension statement at the beginning of the program, with subsequent dimensioning within the subroutines to only the first element of the vector or matrix. This is a convenient feature of the FORTRAN-IV language in which the program is written. The actual vector and matrix sizes are transmitted to the subroutines by an individual parameter list. Construction of the initial dimension statement and core request size is as follows:

The basic size of the program on the IBM-360/67 Digital Computer, without the initial dimension statement, is 272,000 bytes. This figure includes approximately 19,000 bytes of buffer space required for execution. Within the main dimension statement are fifteen variables. However, only three parameters are needed to specify the sizes of these fifteen variables.

Let a= The number of stations along the meridian of the shell times the number of harmonics considered.

Let b= a, plus two fictitious stations times the number

of harmonics considered.

Let c= The number of harmonics considered.

The main dimension statement would then be constructed as,

DIMENSION P(4,4,a), DEE(4,4,a), DST(4,4,a), X(4,a),
PHIXB(a), PHITB(a), Z(4,b), ZO(4,b),
Z2(4,b), Z3(4,b), ZDOT(4,b), IS(99,c),
JS(99,c), ID(99,c), JD(99,c)



The 99's above limit the user to 99 harmonics in any one run and an unlimited number of meridional stations. The core requirement for the general case would be,

272,000 + 216a + 80b + 1584c = bytes of core required.

For a sample calculation of the core requirements consider the example of a spherical cap with 40 stations along the meridian, and an asymmetric analysis with two harmonics. Therefore,

 $a = 40 \text{ (stations)} \times 2 \text{ (harmonics)} = 80$

 $b = 80 + 2 \times 2 \text{ (harmonics)} = 84$

c= 2(harmonics)

Thus, for the variables P, DEE, DST,

 $3 \times (4\times4\times80) = 3840 \text{ (words)} \times 4 = 15,360 \text{ bytes}$

for the variable X,

 $4 \times (80) = 320 \text{ (words)} \times 4 = 1280 \text{ bytes}$

for the varibles PHIXB, PHITB,

 $2 \times (80) = 160 \text{ (words)} \times 4 = 640 \text{ bytes}$

for the varibles Z, ZO, Z2, Z3, ZDOT,

 $5 \times (4x84) = 1680 \text{ (words)} \times 4 = 6720 \text{ bytes}$

lastly, for the varibles ID, JD, IS, JS,

 $4 \times (99 \times 2) = 792 \text{ (words)} \times 4 = 3168 \text{ bytes}$

Therefore, the total size of the main dimension statement would be 27,168 bytes. This figure would be rounded up to the nearest even thousand bytes, i.e. 28,000 bytes. Finally, the core requirement for this example problem would be

272,000 + 28,000 = 300,000 bytes.



III. IMPROVED POLE ROUTINE

SATANS code is based upon Sander's geometrically The nonlinear equations under the conditions of small strains and moderately small rotations. The formulation is in four second order nonlinear partial differential equations in terms of U, V, W, and M, where U, V, and W are the meridional, circumferential and normal displacements respectively, and M is the meridional bending moment. The nonlinear partial differential equations in the coordinates s, O , and t are reduced to uncoupled sets of linear differential equations in s and t by expanding the variables in trigonometric series in the circumferential coordinate Θ_{i} and treating the nonlinear terms as pseudo loads. The first and second derivatives in the meridional coordinate s are replaced by the conventional central finite difference approximations, ie.

$$\{z\}'_{i} = 1/2\Delta \quad (\{z\}_{i+1} - \{z\}_{i-1})$$
 (1)

and

$$\{z\}_{i}^{n} = 1/\Delta^{2} (\{z\}_{i+1} - 2 \{z\}_{i} + \{z\}_{i-1})$$
 (2)

where $\{z\}$ is the vector of U, V, W, and M at the instantion, Δ is the uniform dimension between stations, and primes denote partial derivatives with respect to s. Applying these approximations to the governing set of domain



equations leads to

$$[C]_{i} \{z\}_{i-1} + [B]_{i} \{z\}_{i} + [A]_{i} \{z\}_{i+1} = \{g\}_{i}$$
(3)

When the shell does not have a pole, fictitious stations one increment off of the shell are introduced at each end. Both the governing domain equations and the boundary conditions are applied at the two boundary points. Thus, all finite difference approximations to the derivatives, including those of the boundary conditions, are of order

 Δ^2 . However, prior to the development of SATANS-IIA, the treatment of the conditions to be applied at a pole at either end of a shell was handled by a simple Euler forward or backward difference approximation to the first derivative, with truncation error of order Δ . For example, for a pole at s= 0, where i= 1, the first derivative at the pole was approximated with

$$\{z\}_{1}' = 1/\Delta \quad (\{z\}_{2}' - \{z\}_{1}').$$
 (4)

At the time this procedure for handling the pole conditions was developed (1967) it was thought that this would not significantly alter the solution. However, it has since been discovered that such is not the case.

For the new pole routine, an expanded forward difference approximation of order Δ^2 is used at s= 0 which takes into account the two stations after the pole, instead of just one station after the pole as in the Euler scheme. This approximation is

$$\{z\}'_1 = 1/2\Delta \quad (-3\{z\}_1 + 4\{z\}_2 - \{z\}_3).$$
 (5)



The conditions to be imposed upon the dependent variables at a pole are derived in Reference 14. They are:

For
$$N = 0$$
, $u = v = w' = m' = 0$.

Applying equation (5), these conditions can be put into the matrix form

where the above 3 matrices are DL, DG, and DF within the SATANS programs.

For N= 1,
$$u \pm v = u' = w = m = 0$$
,

where the plus sign applies at an initial pole, and the minus sign at a final pole. The matrix form for these conditions is

For
$$N=2$$
, $u=v=w=m'=0$

the matrix form is

For N > 2,
$$u = v = w = m = 0$$



and DL= identity matrix, DG= DF= null matrices.

The sclution procedure in SATANS is an elimination scheme and starts with

$$\{z\}_1 = -[P]_1\{z\}_2 + \{x\}_1$$
 (6)

where the values in [P] based upon the Euler approximation are defined in Reference 14. The higher order approximation defines a new [P]. This new [P] is obtained by simultaneously solving the pole conditions

[DL]
$$\{z\}_1 + [DG] \{z\}_2 + [DF] \{z\}_3 = \{0\},$$
 (7)

and the domain equation at station 2 next to the pole

$$[C]_{2} \{z\}_{1} + [B]_{2} \{z\}_{2} + [A]_{2} \{z\}_{3} = \{g\}_{2},$$
 (8)

to eliminate $\{z\}$. Thus,

$$\{z\}_{3} = [A]_{2}^{4} (\{g\}_{2} - [C]_{2} \{z\}_{1} - [B]_{2} \{z\}_{2}).$$
 (9)

Substituting equation (9) into equation (7) gives

[DL]
$$\{z\}_1 + [DG] \{z\}_2 + [DF] [A]_2^4 (\{g\}_2 - [C]_2 \{z\}_1 - [B]_2 \{z\}_2) = 0.$$
 (10)

Combining like coefficients of the {z} vector leads to

$$([DL] - [DF] [A]^{4}[C]_{2} \{z\}_{1} + ([DG] - [DF] [A]^{4}[B]_{2})$$

$$\{z\}_{2} = -[DF] [A]^{4}[G\}_{2}.$$

$$(11)$$

Finally, solving for {z} yields



$$\{z\}_{1} = - [DL - DF \times A^{-1}_{2} \times C_{2}]^{-1} [DG - DF \times A^{-1}_{2} \times B_{2}] \{z\}_{2}$$

$$+ [DL - DF \times A^{-1}_{2} \times C_{2}]^{-1} [-DF \times A^{-1}_{2}] \{g\}_{2}.$$

$$(12)$$

Thus, $[P]_1 = -[DL - DF \times A_2^{-1} \times C_2^{-1}][DG - DF \times A_2^{-1} \times B_2]$ and $\{x\}_1 = [DL - DF \times A_2^{-1} \times C_2^{-1}][-DF \times A_2^{-1}][g\}_2$. The new $[P]_1$ matrix has been placed into the "PMATRX" subroutine of SATANS-IIA and the new $\{x\}_1$ vector has been placed in the "FORCE" subroutine.

A listing of the pole routine may be found in Appendix D. To incorporate this new routine into a SATANS-I or-II program, first proceed to the "PMATRX" subroutine and remove the fifteen cards that are between, but not including, "IF(NN.GT.2) GO TO 90" and "11 CONTINUE". These cards are located after statement number "14" and just before statement number "11". Replace the cards removed by the ones listed in Appendix D which read from C IN PMATRX " to "90 M3=MN". Then proceed to the "FORCE" subroutine and remove statement number "10". Replace statement number "10" with the nine cards listed in Appendix D which read from "C IN FORCE" to "DO 11 I= 1,4". Also place " COMMON /IBL5/IBCINL, IBCFNL " into the common area of the "FORCE" subroutine.

This completes the implementation of the new pole routine into either SATANS-I or II.



IV. PROBLEM DESCRIPTION

The geometry of the shallow spherical shell used in this study is identical to that used in Reference 1. Briefly, the shallow shell can be specified by the non-dimensional parameter λ , where

$$\lambda = 2[3(1 - V^2)]^{1/4} (H/h)^{1/2}.$$
 (1)

H is the rise of the shell, h is the thickness, and V is Poisson's ratio. The mass density of the shell is m. All shells analyzed had the following dimensions;

Radii of Curvature $R = R_{\Theta} = 250$ inches

Thickness h = 0.25 inches

Modulus of Elasticity E = 30,000,000 psi

Poisson's Ratio y = 0.3

All buckling pressures obtained will be listed as a percent of the classical buckling pressure of a complete sphere, ${\bf q}_{_{\rm O}}$, where

$$q_0 = [2 E (h/R)^2] / [3 (1 - V^2)]^{1/2}$$
 (2)

Forty stations were used over the meridian. The nondimensional time increment & t, where

$$t = T / (R^2 m / E)^{1/2},$$
 (3)



was taken as 0.05 for 3000 time steps, which is a total nondimensional time of 150. In addition, the axisymmetric analysis was repeated with a larger time step of δ t= 0.2 for a total time of 600. In this study m was selected such that t is equal to T. The necessity for the long response time is explained in Reference 6.

In the axisymmetric analysis only the N= 0 harmonic is considered. However, in the asymmetric analysis a second harmonic is excited by applying an incremental load in that harmonic. In addition, analyses of the shells λ = 6, 7.5, and 11 were made using five harmonics. The step pressure load for the axisymmetric harmonic is

$$\{q^{(0)}\} = P q_0 \{1\},$$
 (4)

and the step pressure load for the asymmetric second harmonic is

$$\{q^{(n)}\} = P q_0 \quad \xi^{(n)} \{1\},$$
 (5)

where n> 0, and ε is taken as 0.0001. The value taken for the second harmonic in the asymmetric analysis was the same as the critical narmonic for the static buckling analysis presented by Stilwell and Ball [2]. When there was an uncertainty as to which was the critical static harmonic the two harmonics in question were both tested. Run times using SATANS-IIA with a two-harmonic analysis for 3000 time steps and 40 stations on the meridian took an average of 28 minutes on the IBM 360/67.

The parameter used to determine the minimum load at which dynamic buckling occurs is the peak value of \bar{v} , called



 $\overline{\mathtt{V}}$, where $\overline{\mathtt{V}}$ is defined as

$$\overline{V} = \int_0^r r W^{(0)} dr / \int_0^r r \xi dr$$
 (6)

r is the normal distance from the axis to the shell, r is the maximum value of r, $\overline{W}^{(0)}$ is the normal displacement of the axisymmetric response and \overline{S} is the vertical distance from the base plane to the undeformed shell. The \overline{V} is a measure of the volume of the shell deformation. The Fortran statements computing \overline{V} and \overline{V} are given in Appendix E.

When working a problem that requires these calculations the nineteen cards are inserted directly into the "DYNAMIC" subroutine right after the "IF" statement that calls the "OUTPUT" subroutine.

For convenience, the response in each asymmetric harmonic is also measured using equation (6), with W (0) replaced with W (n). The parameter \vec{v} for the asymmetric harmonics does not represent a volume of deformation as it does for the axisymmetric harmonic. It can, however, be used to indicate the relative excitation of the asymmetric harmonics.

The buckling criterion for both the axisymmetric and the asymmetric dynamic buckling analysis defines the critical load as that load P where a very small increase in P causes a very large increase in \overline{v} . This is the same criterion MAX



as that used in Ref. [1].

V. RESULTS AND DISCUSSION

A. STATIC AXISYMMETRIC BUCKLING ANALYSIS

I presents the new results from the static Table axisymmetric buckling analyses for $\lambda = 4$ through 13 using the new pole routine. The two upper curves in Figure 1 comparison of the new results obtained SATANS-IIA with those obtained by Stilwell and Ball [2] using the SATANS-I program. As can be seen in this figure, fairly significant changes in the buckling load occurred the neighborhood of $\lambda = 4.5$, and 9; and somewhat smaller differences occurred in the region $\lambda = 10$ through 13. upper data points in Figure 2 present the comparison of the new results from SATANS-IIA with those obtained by This comparison shows a very good agreement between the two sets of results, except for the largest values of λ . The new results have eliminated the differences that existed between the SATANS-I results and Huang's results.

B. DYNAMIC AXISYMMETRIC BUCKLING ANALYSIS

Figure 3 presents the new results for the peak value of $\overline{\mathbf{v}}$ versus P for the various values of λ tested. Table MAX

II presents all of the new results for the dynamic axisymmetric buckling load. These loads are selected from figures constructed just like Figure 3. In every case,



except for λ = 4, a value of P slightly above the P CRIT value caused a \tilde{v} indicative of buckling, as well as a nonconvergence of the iterative solution procedure.

The lower two curves of Figure 1 present a comparison versus λ of the new axisymmetric dynamic buckling results with the previous buckling results obtained by Ball and Burt [1]. In every case the new critical pressure is lower than the critical pressure obtained using the Euler approximation at the pole.

The lower data points of Figure 2 present a comparison of the new results with those obtained by Huang [5], by Stephens and Fulton [6], and by Stricklin [8]. Just as in the case of the static axisymmetric buckling analysis, the new results compare much more favorably with the other results than did the results of Reference 1. It's interesting to note that the new results now tend to be slightly lower than the other results, whereas the results of Reference 1 were higher for almost all values of λ .

C. DYNAMIC ASYMMETRIC BUCKLING ANALYSIS

Table III presents the new results for the critical pressures obtained from the dynamic asymmetric analysis. The second harmonics, or critical static harmonics, used in the analyses are also presented in Table III. A comparison of the critical pressures from the asymmetric analyses, Table III, with the critical pressures from the axisymmetric analyses, Table II, reveals that only the shell λ =6 buckled at a load below the axisymmetric buckling load. For the shell λ = 7 the critical buckling load was slightly



larger when asymmetric motion was considered. In all other cases the buckling was not influenced by the presence of the second harmonic. These new buckling results and those by Ball and Burt [2] are plotted in Figure 4. The new results can be seen to be significantly different from the SATANS-I results, where the asymmetric buckling loads were lower than the axisymmetric loads for five out of the ten values of tested.

Except for $\lambda = 6$ and 7, the relationship between \overline{V} and P for the N= 0 harmonic, in the two-harmonic analyses, was found to be essentially identical to the relationship found in the axisymmetric buckling analysis shown in Figure Table IV A presents the V versus P data for both the 3. N= 0 harmonic and the second harmonic, for all values of λ tested, except for $\lambda = 6$. Note that, except for $\lambda = 7$ and 11, \overline{V} for the asymmetric harmonic is generally very MAX the \overline{V} for the N= 0 harmonic indicates small, even when that the shell has buckled. Thus, except for the shells $\lambda =$ 6 and 7, the presence of the asymmetric motion does not influence the axisymmetric motion, and except for the shells $\lambda = 6$, 7 and 11 the asymmetric motion is very small prior to buckling in the axisymmetric harmonic.

A more detailed analysis of the shell $\lambda=6$ has been conducted since it was the only shell that revealed any significant axisymmetric sensitivity to asymmetric motion. This shell was studied using two two-harmonic analyses (N=0, 1 and N=0, 2) and a five-harmonic analysis (N=0, 1, 2, 3, and 4). Figure 5 and Tables IV B and IV C contain values of \overline{V} versus P for both of the asymmetric harmonics, N= 1 MAX



and N= 2, in the two two-harmonic analyses, as well as the values of \overline{V} for the axisymmetric harmonic, N= 0. Figure 6 and Table IV D present the values of $\overline{\mathtt{V}}$ versus P for the MAX N= 0,1,2,3, and 4 harmonics from the five-harmonic study. A comparison of the critical buckling load predicted from the results of the two two-harmonic analyses in Figure 5 with the critical load from the five-harmonic analysis obtained from Figure 6 shows that the presence of the additional harmonics results in the shell buckling at a slightly lower load (0.50), with significant motion in the N= 1 harmonic instead of the N= 2 harmonic (see the norconverged solution at P= 0.51), which is the critical harmonic for static asymmetric buckling. Studies using five harmonics have also been conducted for $\lambda = 7.5$ and $\lambda = 11$. As can be seen in Table IV D the critical harmonic for $\lambda = 7.5$ remained N= 3; however, significant motion occurred in that harmonic at P= .41 and .44. In the case of $\lambda = 11$, relatively large asymmetric motion occurred in the asymmetric mode of N= 5 vice 6 at a value of P= .46.

The comparison of the new results for the critical pressure for dynamic asymmetric buckling with those obtained analytically by Stricklin [8], by Akkas [9], and experimentally by Lock et al [7] is illustrated in Figure 7. The comparison reveals an agreement with Stricklin in every case, in general a higher value of P than those obtained CRIT

by Akkas, and most importantly a very good agreement with Lock's experimental results.

When making the comparison between the new results and those obtained by Akkas, it is necessary to look at the differences in the problem solution parameters used in the two studies. For example, buckling results obtained from SATANS-IIA using the same time increment as used by Akkas,



 δ t= .2 for 3000 time steps, were significantly higher than those using the time step of δ t= .05 for many values of λ . Furthermore, the new results had, in some cases, instances of buckling occurring as far out in time as 130. Akkas, to shorten computer run times, observed the cap only for a time of less than 5. Furthermore, only the harmonics N= 1 or 2 or 3 were studied by Akkas for shells λ = 5 through 12. If the critical harmonic is not studied, the predicted load will be too high. Thus, it appears that Akkas' lower bound loads may not be true lower bounds.

Two additional features of the shell response should be noted. First, shells $\lambda = 6$, 7.5, and 11 exhibited a non-buckled response in the axisymmetric harmonic to a load larger than the defined critical buckling load. This can be seen in Tables IV A and IV C. Second, and most importantly, the buckling load proposed by Ball and Burt [1], and used here, defines buckling to occur when the in axisymmetric harmonic undergoes a large change due to a small change in P. Another criterion for dynamic buckling in the asymmetric analysis discussed in Reference 1 is to define the buckling load as that threshold load that initiates significant growth in the asymmetric harmonic. Re-examination of the \overline{V} versus P data in Table through D reveals that shells $\lambda = 6$, 7, and 11 exhibited relatively large asymmetric motion at loads smaller that the defined buckling load when compared with other $\overline{\mathtt{V}}$ for those shells, even though the numbers themselves were small when compared with the axisymmetric harmonic. Shells? =7.5 and 12 appear to be borderline cases. If the alternate criterion for buckling is used, the critical buckling loads for shells $\lambda = 6$, 7, and 11 become 0.47, 0.45, and 0.45, respectively. The shells $\lambda = 7.5$ and 12 could have buckling



loads as low as 0.40 and 0.44, respectively. These values are more conservative than the definition based upon axisymmetric response. These five shells are the same five shells that exhibited an asymmetric buckling load lower than the axisymmetric buckling load in Reference 1.



VI. SUMMARY AND CONCLUSIONS

digital computer program for the geometrically nonlinear analysis of totally arbitrarily loaded shells of modified to (SATANS-II) was more accurately account for the conditions at the pole of the shell. program, called SATANS-IIA, was used to determine the buckling load of shallow spherical shells of various when subjected to static axisymmetric, dynamic axisymmetric, and dynamic nearly axisymmetric step-pressure loads of The cap sizes ranged from $\lambda = 4$ to 13 infinite duration. including $\lambda = 7.5$. A comparison was made between the new buckling results with the improved pole handling routine and the results that did not have the new pole routine. The revealed a significant change in buckling comparison pressures, due solely to the change from an order Δ difference approximation of the first derivatives at the pole to an approximation of order Δ . These new critical

pressures are in very good agreement with the results from other studies of the same spherical shells. This good agreement with other results, which came about as a result of the modification of the pole handling routine, is a strong indication that the manner in which the pole condition is handled is vital to the accuracy of the solutions obtained.

In the asymmetric analysis, two harmonics were included for most of the shells; the axisymmetric harmonic and one asymmetric harmonic. Five-harmonic analyses were conducted for three of the shells. Two buckling criteria for the



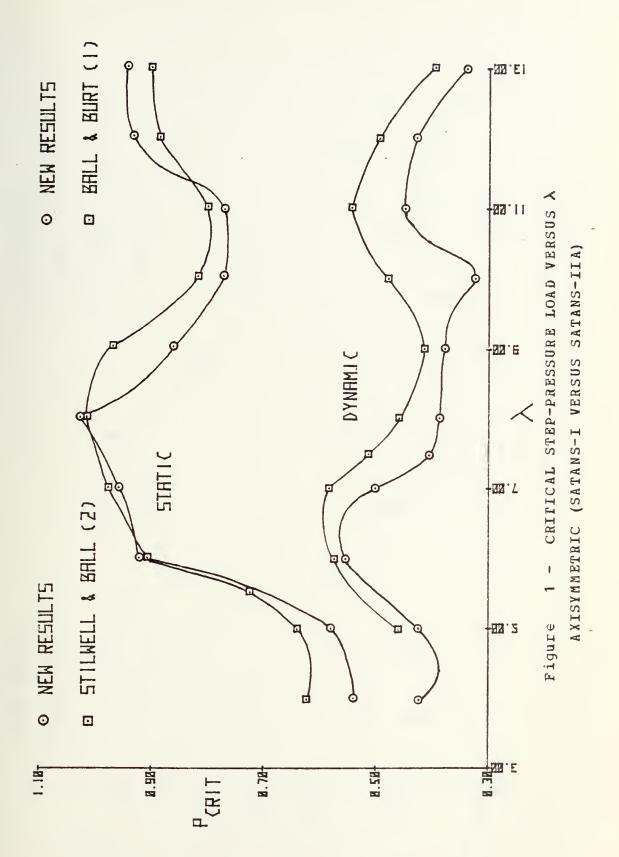
asymmetric analysis were considered. One defined buckling as that threshold load that caused a large increase in a deformation parameter, \tilde{v} , in the axisymmetric harmonic.

The other, more conservative than the first, defined buckling as that threshold load that caused a large increase in the \bar{v} value for the asymmetric harmonic. Both values

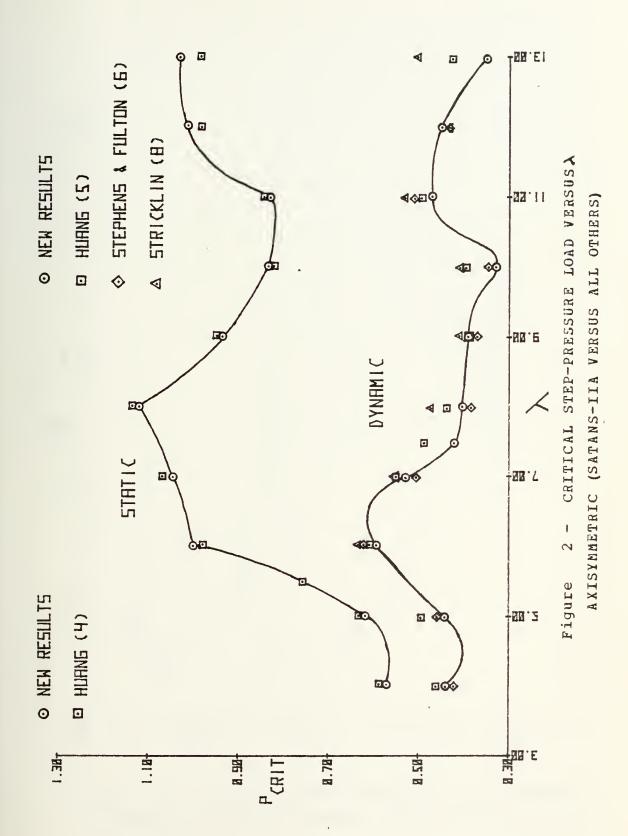
have been presented.

The new static axisymmetric, dynamic axisymmetric, and even the dynamic asymmetric critical buckling pressure loads appear to be fairly reliable results for perfect, shallow shells. The effect of realistic imperfections remains to be determined.











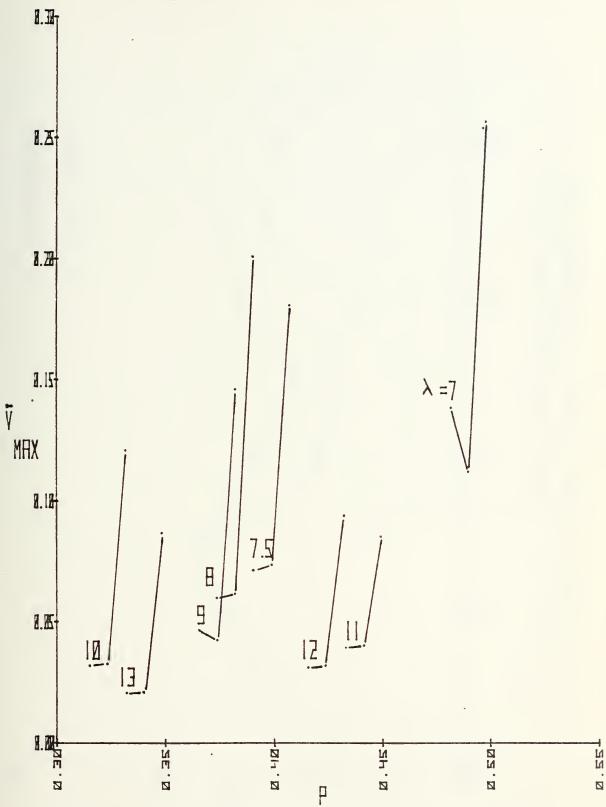
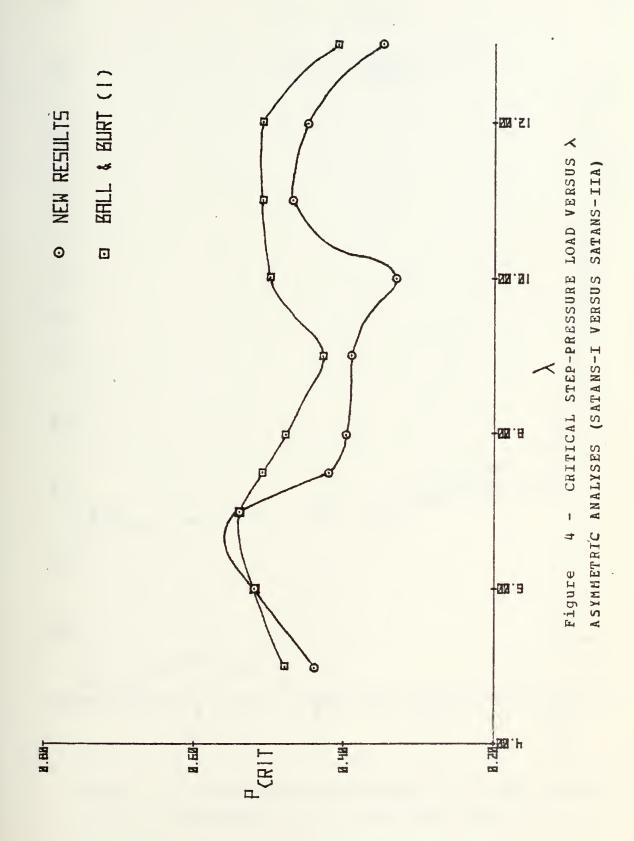


Figure 3 - PEAK DEFLECTION VERSUS P, AXISYMMETRIC AND ASYMMETRIC CASES FOR VARIOUS VALUES OF λ (SATANS-IIA)







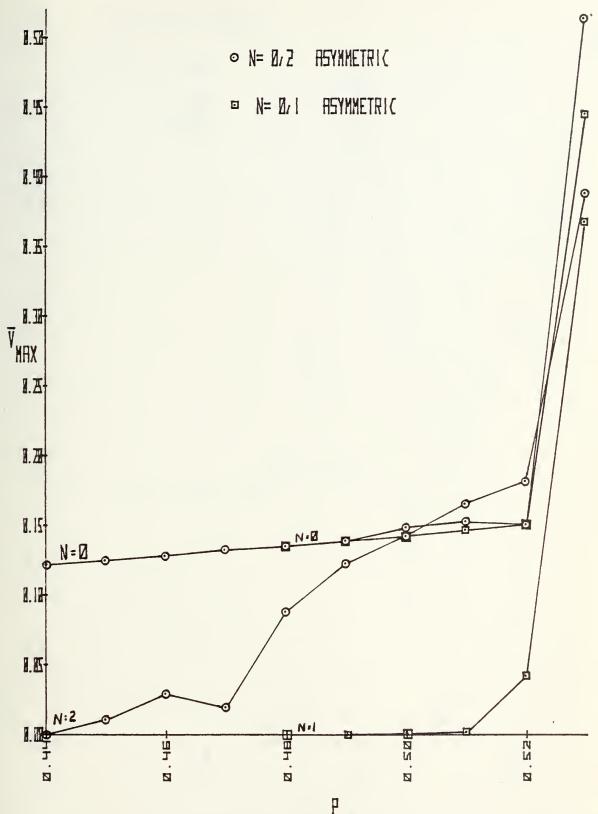
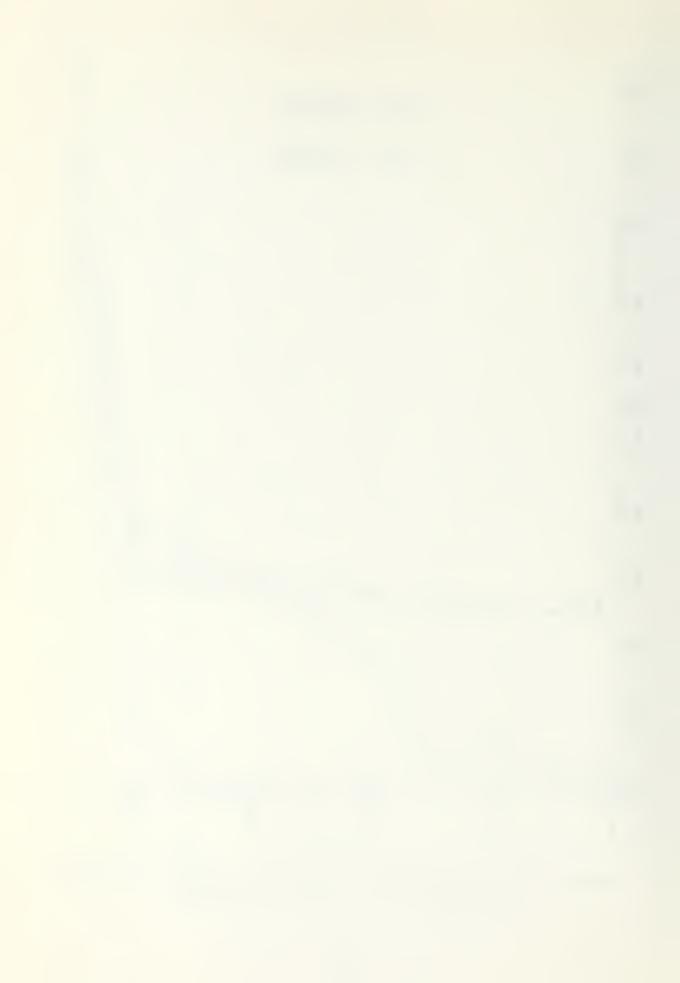


Figure 5 - PEAK DEFLECTION VERSUS P FOR THE ASYMMETRIC ANALYSES OF λ = 6 (N=0,1 AND N=0,2)



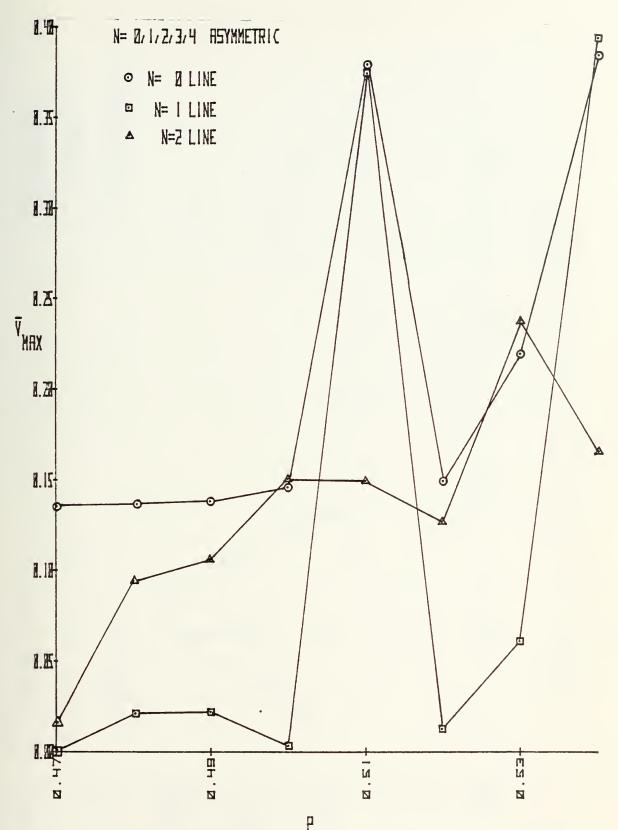
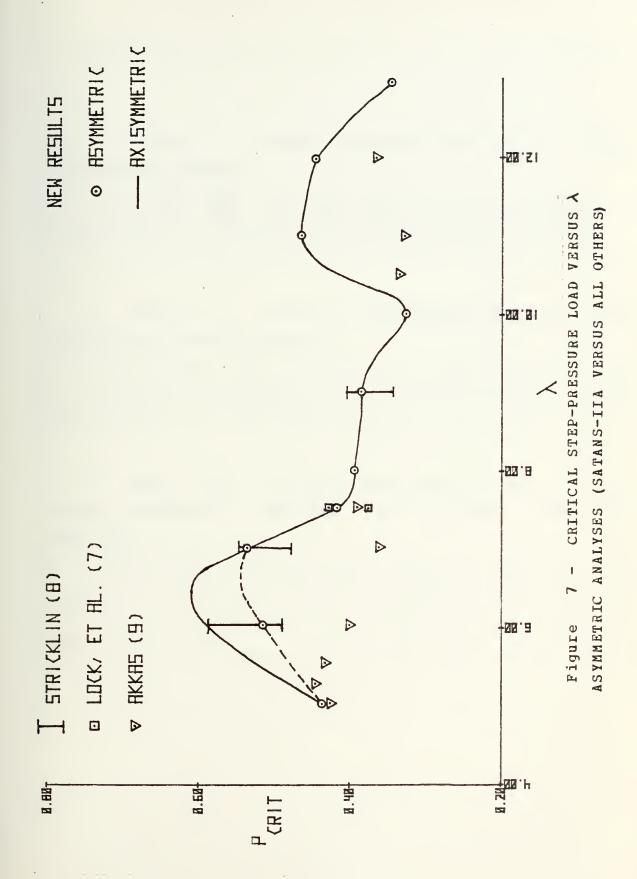


Figure 6 - PEAK DEFLECTION VERSUS P FOR THE ASYMMETRIC ANALYSES OF λ = 6 (N=0,1,2,3,AND4, ONLY N=0,1,AND2 PLOTTED)







A. TABLES

1. TABLE I Critical pressure loads from the static axisymmetric analyses.

λ	4	5	6	7	8	9	10	11	12	13
PCRIT	.568	.616	1.0	1.048	1.12	.936	. 832	.832	1.016	1.032

2. TABLE II Critical step-pressure loads from the axisymmetric dynamic analyses.

λ	4	5	б	7	7.5	8	9	10	11	12	13
PCRIT	. 45	-44	.59	.53	.42	.40	.39	.33	. 47	.45	.35

3. TABLE III Critical step-pressure loads from the dynamic asymmetric analyses and critical asymmetric harmonics.

λ	5	6	7	7.5	8	9	10	11	12	13
PCRIT	- 44	.52	- 54	. 42	-40	. 39	.33	-47	.45	.35
NCRIT	1	2	3	3	4	5	6	7	8	9



- 4. TABLE IV Dynamic asymmetric analyses for VMAX versus <u>P.</u>
 - 1. TABLE IV A. Two-harmonic analyses for all values of λ exept $\lambda = 6$.

 $\lambda = 5 N = 0 \text{ and } 2$

N=0 and 1

_	 					
P	.43	. 44	.45	P	.44	. 45
N = 0	. 1659	. 1676	.6606	N = 0	. 1675	.6606
N= 2	.0004787	.0000566	.0687	N= 1	.0003145	.0687
				P	.46	
				N = 0	.7 653	
				N= 1	.001092	2

 $\lambda = 7$, N= 0 and 3

_^		. ,						
P		.45	.46	.47	.48	.49	.50	. 52
N =	0	.09452	.09571	.09812	.1005	.1029	.1052	.1099
N =	3	.000889	.007456	.05052	.04323	.0279	.0335	.07488
P		.53	. 54	.55				
N =	0	.1122	.1146	.2709]			
N=	3	. 05997	06252	.03809				

 $\lambda = 7.5$, N= 0 and 3

P		-40	.41	.42	.43	.44	.45
N =	0	.0703	.07228	.07429	.2636	.07837	. 2076
N=	3	.0001094	.001296	.0004304	.002754	.001188	.000338
P		. 46					
И=	0	. 200					
N=	3	.000327	16				



	λ	=	8,	N=	0	and	4
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P	.38	. 39	.40	.41	.42	.43
N = 0	.05893	.0607	.0624	.1964	. 1713	. 1957
N= 4	.0000566	.0000703	.0000364	.0000333	.0000274	.0000299
Р	. 44					

P .44 N= 0 .2297 N= 4 .0000326

 $\lambda = 9 N = 0 \text{ and } 4$

N = 0	and	5
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P	.38	.39	.40	P	.40
N = 0	.04738	.04875	.1576	N = 0	.05012
N= 4	.00003597	.00004635	.00004497	N = 5	.00008385

 $\lambda = 10$, N= 0 and 5

P	.32	.33	.34	. 36	.38	.40
N = 0	.03239	.03347	.1086	.1217	.1288	. 1235
N= 5	.0000281	.0000472	.00004125	.00002103	.0000449	.000114

 $\lambda = 11$, N= 0 and 6

P	. 45	. 46	.46	.48	.49	. 50
N = 0	.03910	.04004	.04099	.09814	.04241	.08824
N= 6	.004595	.01332	.02232	.02864	.03955	.02813

 λ = 12, N= 0 and 7

P	. 44	. 45	.46	
N = 0	.03236	.03316	.08633	
N= 7	.00004214	.0004561	.00005158	

 λ = 13, N= 0 and 8

P	.34	. 35	.36	.38	. 40
N = 0	.02119	.02185	.06637	.07844	.07381
N= 8	.00001148	.00001134	.000006607	.000008245	.000119



2. TABLE IV B. Two-harmonic analyses with N= 0 and 1, λ = 6 cnly.

P		.48	.49	.50	.51	.52	.53
N =	0	. 1350	.1385	.1421	. 1460	. 1499	.4453
N=	1	.0002797	.000195	.000245	.000926	.04081	.3668

3. TABLE IV C. Two-harmonic analyses with N= 0 and 2, λ = 6 only.

P		.44	. 45	.46		.47		.48		49	.50
N =	0	.1218	. 1250	.1276	•	1320	. 1.	350	. 13	85	.1479
N =	2	.000239	.0101	.0293	•	01976	.0	8768	. 12	23	.1419
P		.51	.52	.53		.54		.55)	•	56
N =	0	.1526	.1499	.513	7	.506	0	.20	40	•	5305
N=	2	.1654	.1816	.3878	3	.399	6	. 21	56	•	3617

4. TABLE IV D Five-harmonic analyses for selected shells.

 $\lambda = 6 N = 0, 1, 2, 3, \text{ and } 4$

.05298

P		. 47	.48	.49	.50	.51	.52	.53
N=	0	. 13 13	. 1347	.1382	.1460	.3797	.1498	. 2200
N =	1	.00021	.02108	.02215	.003676	.3743	.01276	.0616
И=	2	.0187	.0953	.1069	.1507	.1502	.1279	. 2385
N=	3	.000181	.006237	.01437	.00163	. 04 05	.0123	.03978
N =	4	.0031	.04757	.05428	.04402	.05896	.0495	.064
P		.54						
N=	0	.3854						
N=	1	.3953						
N=	2	1671						



 $\lambda = 7.5 N = 0, 1, 2, 3, 4$ and 4

P		• 40	.41	. 42	.43	.44	.45
N =	0	.0703	.07228	.07429	. 2592	.07837	. 2544
N=	1	.00004855	.00004198	.00006093	.0002737	.0001167	.005952
N=	2	.0001164	.00007456	.0004184	.0000982	.000788	.0003188
N=	3	.0001277	.001187	.0004597	.0002853	.00107	.0003188
N=	4	.0008224	.0001898	.0002448	.0000526	.000280	.000134

 $\lambda = 11 \text{ N} = 0,4,5,6, \text{ and } 7$

P		•45	.46	.47	.48	.49	.50
N=	0	.03910	.04004	.0499	.04195	.04291	.1040
N=	4	.0005759	.001263	.0009774	.001388	.002657	.001565
И=	5	.009568	.0140	.0124	.02239	.02759	.01548
N =	6	.002560	.007828	.02330	.02767	.02602	.02644
N =	7	.0001743	.0001486	.0002021	.01202	.02048	.02064



APPENDIX A

LISTING OF SATANS-IIA



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IF (K. EQ. *K AX. AND. IBCFNL. LT. 0) GO TC 60 CALL POLE (K. P. CELL PHIBET (K. Z. IS. JS. ID. JD. PHIXB)

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RITE (6,1000) F (INSTH-GT.0) CALL PLCTIT (XSTATN, YNSTH, KM, FITE (6,2003) F (ICS.Eq.0) GO TO 18 FITE (6,1000)	F (105.61.0) CALL PLCIII (XSIAIN,YUS,KMAX,C F (105.LT.0) CALL PLCIII (XSTAIN,YQS,NGKMAX RITE (6,2004) F (1MS.EQ.0) GO TO 19 RITE (6,1000)	IMS.GT.0) IMS.LT.0) E (6,2005) IMTH.EQ.0 E (6,1000) IMTH.GT.0	FITE (6, 2006 RITE (6, 1000 F (IMSTH-GT- F (IMSTH-GT- F (IMSTH-CT-	1	HRITE (61000) IF (IV. 61.00) IF (IV. LT.0) CALL PL HRITE (6,209) 4 IF (1W. 60.0) GO TO 2	F (IM.GT.0) CALL PLOTIT (XSTATN, YW, KMAX, 0) F (IM.LT.0) CALL PLOTIT (XSTATN, YW, NGKMAX, C) RITE (6,2008) F (IPHIS.EQ.0) GO TO 26 FITE (6,1000) F (IPHIS.GT.0) CALL PLOTIT (XSTATN, YPHIS, KPAX, 0)	f (IPHIS.LT.0) CALL PLCTIT (XSTATN,YPHIS,NGKWAX FITE (6,2011) f (IPHIT.EQ.0) GO TO 27 FITE (6,1000) f (IPHIT.GT.0) CALL PLCTIT (XSTATN,YPHIT,KWAX,O
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, DETERM, IPIVOT, INCEX, 4, ISCALE
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AAIINV(A,4,61,0,DETERM,IPIVGT,INCEX,4,ISCALE)

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29 KN) = 1.

(MI) • L T • 0) DL (2,2, MN) = -1.

49 KN) = 1.

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19 KN) = 4.
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CC 44 L=1,4
SCNOP=SCMOP+PATA(I
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ENR*D1*BXTZT)+CBS*(BXZT+BEZT+NU*(B1ZT+BEZT))-2.*CX*

(EXXZT+ETXZT)-ENR*(EXZT+ETZT))*TOEL

E(2)+OSE*(BS*(ENR*(BTZT+BEZT+NU*(BXZT+BEZT))-D1*

(T+2.*GA*BXTZT))-D1*OBS*BXTZT+2.*CT*(ETTZT+EXTZT)

(DEX+DET))*TOEL

E(3)+OSE*(BS*((OX+NU*OT)*(BXZT+BEZT)+(OT+NU*OX)*

BTZT+BEZT))+2.*(GA*(EXXZT+ETXZT)+CEXX+DETX+ENR*

EXTZT+ETTZT))*TOEL
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S(IX=SUPX+CL(II,L,P)*GEE(L)

X(II,IKI)=SUMX

CCNTINUE

CC 11 I=1,4

SLPX=0.

CC 12 J=1,4

SLPX=SUMX+DEE(I,J,IK)*GEE(J)

X(I,IK)=SUMX

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I F (ABS (DETERM)-RI) 1030,1010,1010

CETERM = DETERM/RI

I SCALE = 1 SCALE + 1

I F (ABS (DETERM)-RI) 1060,1020,1020

CETERM = DETERM/RI

I SCALE = 1 SCALE + 1

I CALE = 1 SCALE - 1

I F (ABS (DETERM*RI) - R2) 1050,1050,1060

CETERM = DETERM*RI

I SCALE = 1 SCALE - 1

I F (ABS (PIVOTI) - R1) 109C,1070,1070

I SCALE = 1 SCALE + 1

I F (ABS (PIVOTI) - R1) 320,1080,1080

FIVCTI = FIVCTI/RI

I F (ABS (PIVOTI) - R1) 320,1080,1080
                                                                                                                                             COLUM) = IP IVCT (ICOLUM) +1
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CC 200 L=1,N

Shap=A(IROh,L)

A(IROW,L)=A(ICGLUM,L)

A(ICCLUM,L)=SWAP

IF (M) 260, 260, 210

CC 200 L=1, M

Shap=B(IROW,L)

E(IROW,L)=B(ICGLUM,L)

E(ICGLUM,L)=SWAP

INCEX(I'1)=IROW

INCEX(I'1)=IROW

INCEX(I'2)=ICGLUM
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550 L1=1,N (L1-1COLUM) 400 A(L1,1COLUM) L1,1CCLUM)=0.0

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ISCALE=ISCALE+1
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IF (ABS(PIVOTI)-R2)20CC,2000,320
PIVCTI=PIVCTI*RI
ISCALE=ISCALE-1
IF (ABS(PIVOTI)-R2)2010,201C,320
PIVCTI=FIVCTI*RI
ISCALE=ISCALE-1
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CALL BCE(1, 8, DB, D, DC)

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CALL BCB(K,B,DB,D,DD)

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(1.2) = NU*8*ENR

(1.3) = B*( 0X+NU*CT)

(1.4) = 0.

(2.1) = -B*D1*ENR / 2. -DL / 8.* 0XT*CTX*REG

(2.3) = -GA*H(2,2)

(2.3) = -GA*H(2,3)

(2.3) = -L2*D*D1*((1.+NU)*GA2*GX+ENR2/4.3)

(3.3) = -L2*D*D1*((1.+NU)*GA2*GX+ENR2/4.3)

(4.2) = 0.

(4.3) = 0.

(4.4) = 0.

(4.5) = 0.

(4.5) = 0.
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2•*CT*(1•+NU)+GT
+NC+YAH)*GA*ENR2
 =L2*D*D1*(YAH*ENR2+(1.+NU)*GA**2)
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LAMAXI, KMAXZ, NCCNV

ZFIM(4,4,59), ZFZW(4,4,59), ZFZW(4,4,59), ZFZW(4,4,59), ZFZW(4,4,59), ZFZW(4,4,59), ZFZW(4,4,59), PHE (50), PHE (50
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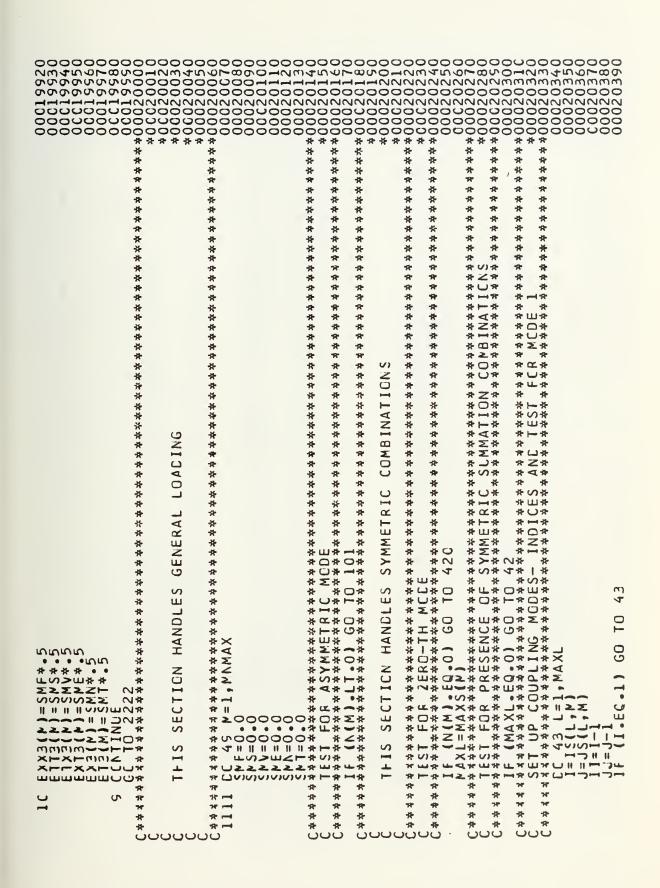
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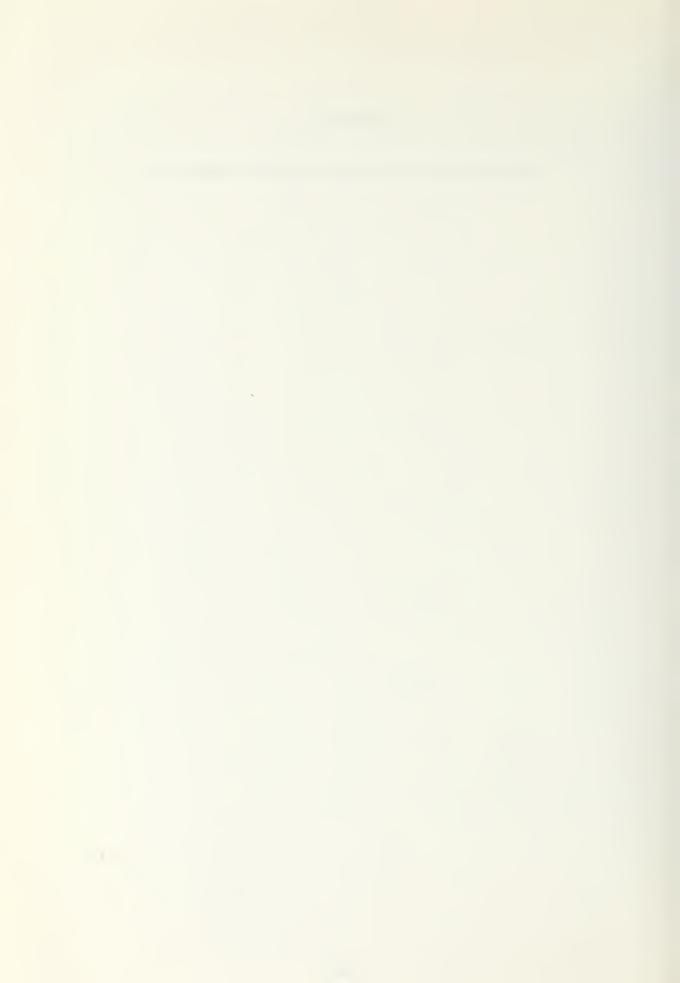
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APPENDIX B

LISTING OF OUTPUT FROM EXAMPLE PROBLEM



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APPENDIX C

INPUT DATA GUIDE FOR SATANS-IIA

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ATT-SHAPES FOR SOTTED ATAI THERET

INPUT DATA GUIDE FOR SATANS-I, SATANS-II, AND SATANS-IIA

CARE	CELLMAS	FORMAT	ITEM	EXAMPLE	MEANING
1	1-72	18A4	TITLE	-	ENTER ANY 72 CHARACTERS
2	1-5	15	NO	1	THE PROBLEM NUMBER, O <nc<10000.< td=""></nc<10000.<>
2	6-10	L5	\$DYNAMC	F	FOR A STATIC ANALYSIS, SET \$DYNAMC = F. FOR A DYNAMIC ANALYSIS, SET \$DYNAMC = T.
2	11-15	15	IMODE	0	FOR NO MODAL CUTPUT DATA FOR MCDAL GUTFUT DATA.
2	16-20	I 5	NDIMEN	0	DIMENSIONAL CUTPUT DATA. NONDIMENSIONAL CUTPUT.
2	21-25	I 5	NTHMAX	8	SUMMED SOLUTION WILL BE PRINTED AT NTHMAX MERIO-IANS, O<=NTHMAX<=36.
2	26-30	15	IFREQ	2	SCLUTION WILL BE PRINTED AT THE FIRST STATION, EVERY SUBSEQUENT IFREC STATION AND THE LAST STATION, O IFREC< KMAX.</td
2	31-35	I 5	IPRINT	3	EVERY IPRINT CONVERGED SOLUTION WILL BE PRINT-ED.
2	36-40	15	IBCINL	-1 0	IF THE SHELL HAS A POLE AT THE FIRST STATION. IF THE SHELL HAS NO POLE AT THE FIRST STATION.
2	41-45	I 5	IBCFNL	-1 0	IF THE SHELL HAS A POLE AT THE LAST STATION. IF THE SHELL HAS NO POLE AT THE LAST STATION.



CARE	COLUMNS	FORMAT	ITEM	EXAMPLE	MEANING
2	46-50	15	KMAX	35	NUMBER OF MERICICNAL STATIONS. NOTE: KMAX<201 FCR SATANS -I WITHOUT PLCTS AND KMAX<101 FOR SATANS-I WITH PLOTS OR FOR SATANS -II. SATANS-IIA IS UNLIMITED.
2	51-55	I 5	MNMAX	7	NUMBER OF SERIES COEFFI- CIENTS USED TO DESCRIBE THE INITIAL CONDITIONS, PRESSURE AND THERMAL LOADS (AND INITIAL IMPER- FECTIONS IF USING SATANS -II OR IIA). MNMAX<=MAXM.
2	56-6C	15	MAXM	7	MAX NUMBER OF FARMONICS IN THE SOLUTION, LIMITED TO SS.
2	61-65	I 5	LSMAX	s9 3000	FOR A LINEAR ANALYSIS. USE MANY LOAD STEPS FOR A NONLINEAR STATIC ANALYSIS. FOR A DYNAMIC ANALYSIS, LSMAX IS THE NUMBER OF TIME INCREMENTS, WHERE LSMAX = T MAX/AT.
2	66-70	15	LCHMAX	0	THE NUMBER OF LOAD STEP SIZE REDUCTIONS IN A STATIC ANALYSIS, RECOM-MENDED RANGE = 2-4. FOR A DYNAMIC ANALYSIS.
2	71-75	15	ITRMAX	30	FOR A LINEAR ANALYSIS. THE NUMBER GF ITERATIONS AT A LOAD OR TIME STEP. FOR A NONLINEAR ANALYSIS, SUGGESTED RANGE = 10-30, UP TC 50 FOR SPECIAL CASES.
2	76-EC	15	IC	0	INITIAL CONCITIONS. SET TO G FOR A STATIC ANALY- SIS, OR FOR A EYNAMIC ANALYSIS WHERE THE SHELL IS AT REST AT T= 0. FOR A CYNAMIC ANALYSIS WITH INITIAL CONCITIONS.



CARE	CCLUMN	S FORMAT	ITEM	EXAMPLE	MEANING
3	1-12	E12.3	NU	0.3	POISSON'S RATIO, V.
3	12-24	E12.3	SIGO	1000.0	REFERENCE STRESS LEVEL,
				1.0	IF THE INPUT DATA IS DIMENSIONAL.
3	24-36	E12.3	ELAST	•3E8	REFERENCE MCCULUS OF
				1.0	ELASTICITY, E IF THE INPUT DATA IS DIMENSIONAL.
3	37-48	E12.3	TKN	•4E-2 1.0	REFERENCE THICKNESS, h IF THE INPUT DATA IS
				1.0	DIMENSIONAL.
3	49-6C	E12.3	CHAR	8.16	CHARACTERISTIC SHELL
				1.0	DIMENSION, A . IF THE INPUT DATA IS DIMENSIONAL.
3	61-72	E12.3	TEEO	0.0 .996E-5	IF A STATIC ANALYSIS.
				•996E-5	REFERÊNCE TIME, To.
4	1-12	F12-3	DELCA	D 0.2	FOR A STATIC ANALYSIS.
		21243		5 002	FOR A STATIC ANALYSIS, DELCAD IS THE LCAD INCRE- MENT. IT REMAINS UN- CHANGED UNTIL THE SOLU- TION FAILS TO CONVERGE IN ITERMAX ITERATIONS, WHEN IT IS REDUCED BY A FACTOR
					CHANGED UNTIL THE SOLU- TION FAILS TO CONVERGE IN
					ITERMAX ITERATIONS, WHEN IT IS REDUCED BY A FACTOR
					LCHMAX SUCH RECUCTIONS
				.1823E-6	WILL CCCUR. FOR A DYNAMIC ANALYSIS,
					FOR A DYNAMIC ANALYSIS, DELGAD IS THE NONDIMEN- SICNAL TIME INCREMENT.
4	13-24	E12.3	EPS	0.01	THE CONVERGENCE CRITERION RECOMMENDED RANGE OF
					0.01 <eps<0.001.< td=""></eps<0.001.<>
				500	TANC 11 00 047440 114 000
					TANS-II OR SATANS-IIA RUN.
4 A	1-5	15	JUMP	1	FOR AN ANALYSIS USING SINGLE SERIES EXPANSIONS. FOR AN ANALYSIS USING DOUBLE SERIES EXPANSIONS.
				2	FOR AN ANALYSIS USING DOUBLE SERIES EXPANSIONS.
4 A	5-10	15	MPERF	s 0	AN ANALYSIS WITHOUT IM-
				1	PERFECTIONS. AN ANALYSIS WITH IMPERFEC-
					TIONS. NOTE: IF JUMF=28 MPERFS
					MAY BE O OR 1. IF JUMP =1, MPERFS MUST BE O. IF MPERFS=1, JUMP MUST BE 2.
					MPERFS=1, JUMP MUST BE 2.



CARC COLUMN FORMAT ITEM EXAMPLE MEANING

INCLUDE AS MANY CARDS 5 AS NECESSARY TO SPECIFY NTHMAX MERICIANS. IF NTHMAX EQUALS 0, OMIT CARD 5.

6E12.3 5 1-72

10.0

A LIST OF CIRCUMFERENTIAL COORDINATES # , IN DEGREES AND TENTHS, WHERE THE SCLUTION PRINTCUT IS CE-SIRED. THE LIST MUST HAVE NTHMAX ENTRIES.

IF IBCINL= -1, CMIT CARDS 6 THROUGH 14. IF IBCFNL= -1, CMIT CARCS 15 THROUGH 23. CARDS 6 THROUGH 23 DESCRIBE THE BCUNCARY CONDITIONS AT THE FIRST, AND THEN AT THE LAST STATION. THE BOUNDARY CONDITIONS EXIST ON THE TOTAL VARIABLES, NOT ON THE INDIVIDUAL HARMONICS. LOADINGS APPLIED THROUGH SPECIFICATION OF BOUNDARY CONDITIONS ARE TAKEN IN THE ZERCETH HARMONIC (N=0) ONLY, AS THE COLUMN MATRIX [%] IS SET TO ZERC FOR HARMONICS GREATER THAN ZERO. THE BOUNDARY CONDITIONS ARE DIMENSIONAL. THE FORMAT OF CARDS 6 THROUGH 23 4E16.8.

$$\begin{bmatrix} \Omega(1,1) & \Omega(1,2) & \Omega(1,3) & \Omega(1,4) \\ \Omega(2,1) & \Omega(2,2) & \Omega(2,3) & \Omega(2,4) \\ \Omega(3,1) & \Omega(3,2) & \Omega(3,3) & \Omega(3,4) \\ \Omega(4,1) & \Omega(4,2) & \Omega(4,3) & \Omega(4,4) \end{bmatrix} \begin{bmatrix} N_s \\ N_{so} \\ Q_s \\ P_s \end{bmatrix}$$

CARC 10,19 CARD 11,20 CARD 12,21 CARC 13,22

[
$$\Lambda$$
(1,1) Λ (1,2) Λ (1,3) Λ (1,4)

[Λ (2,1) Λ (2,2) Λ (2,3) Λ (2,4)

[Λ (3,1) Λ (3,2) Λ (3,3) Λ (3,4)

[Λ (4,1) Λ (4,2) Λ (4,3) Λ (4,4)

CARC 24 IS:

1. INCLUDED FOR A SATANS-I STATIC ANALYSIS.

2. INCLUDED BUT BLANK FOR A SATANS-I DYNAMIC ANALYSIS.

3. CMITTED FOR A SATANS-II ANALYSIS.

4. INCLUDED BLANK FOR DYNAMIC USED FOR STATIC SATANS-IIA ANALYSES.

CARE	CCLUMN	FCRMAT	ITEM	EXAME	PLE	MEANING
24	1-2	L2	\$PLOT	S	F T	INDICATES PLCTS ARE NCT DESIRED. INCICATES PLCTS ARE DESIRED:
24	3-4	L2	\$MCDA	L	F T	INCICATES PLCTS ARE FCR SUMMED SOLUTIONS ONLY. INCICATES PLCTS ARE FCR



FOR THE REMAINDER OF CARD 24 ENTRIES, C INDICATES THAT NO PLOTS ARE DESIRED FOR THE PARTICULAR ITEM, AND 1 INCICATES THAT THEY ARE DESIRED. ALL GRAPHS ARE PLOTTED AS THE INCICATED ITEM VERSUS THE STATION NUMBER. IF A COMPLETE PLOT IS DESIRED, INSUTE IFREQ = 1.

CARE	CCLUMN	FCRMAT	ITEM EXA	MPLE	MEANING
24	5- 6	12	IRADII	1	PLCT THE RACII AS COMPUT- ED BY SUBROUTINE GEOM.
24	7- 8	12	IGAMMA	1	PLCT P'/P AS CEMPUTED BY Subroutine geom.
24	9 - 10	I 2	IOMEGS	1	PLCT ω_s as computed by subroutine geom.
24	11-12	I 2	IOMEGT	1	PLCT ωe as computed by subroutine geom.
24	13-14	I 2	IDECMS	1	PLCT ω_s as computed by subroutine geom.
24	15-16	Ι2	IBSTIF	1	PLCT THE STIFFNESS C AS COMPUTED BY SUBROUTINE BDE.
24	17-18	12	IDSTIF	1	PLOT THE STIFFNESS D AS COMPUTED BY THE SUBROUTINE BDB.
24	19-20	I 2	IBBSTF	1	PLCT THE STIFFNESS 46/45 AS COMPUTED BY SUBROUTINE BDB.
24	21-22	12	IDDSTF	1	PLCT THE STIFFNESS dd/d\$ AS COMPUTED BY SUBROU-
24	23-24	12	IPR	1	TINE BDB. PLOT THE NORMAL COMPONENT OF THE PRESSURE LOAD.
24	25-26	12	IPS	1	PLOT THE MERIDIONAL CCM- PCNENT OF THE FRESSURE LOAC.
24	27-28	Ι2	IPT	1	PLCT THE CIRCUMFERENTIAL COMPONENT OF THE PRESSURE LOAD.
24	29-20	I 2	ITT	1	PLCT THE THERMAL LOAD.
24	31-32	12	IMT	1	PLOT THE THERMAL MOMENT.
24	33-34	12	IDTT	1	PLCT d/df OF THE THERMAL LCAC.
24	35-36	12	IDMT	1	PLOT d/df OF THE THERMAL MOMENT.
24	37-38	12	INS	1	PLOT THE MERIDIONAL MEM- BRANE FORCE DISTRIBUTION.



CARE	CCLUMN	FCRMAT	ITEM	EXAMPLE	MEANING
24	39-4C	12	INTH	1	PLCT THE CIRCUMFERENTIAL MEMBRANE FORCE DISTRIBU-TION.
24	41-42	12	INST	1	PLCT THE MERIDIO- CIRCUMFERENTIAL MEMBRANE FORCE DISTRIBUTION.
24	43-44	I 2	IQS	1	PLCT THE TRANSVERSE FCRCE DISTRIBUTION.
24	45-46	12	IMS	1	PLCT THE MERIDIONAL MCM- ENT DISTRIBUTION.
24	47-48	12	IMTH	1	PLCT THE CIRCUMFERENTIAL
-24	49-50	Ι2	IMST	1	MOMENT DISTRIBUTION. PLCT THE MERICIC- CIRCUMFERENTIAL MOMENT DISTRIBUTION.
24	51-52	12	IU	1	PLCT THE MERIDIONAL CIS- PLACEMENT DISTRIBUTION.
24	53-54	12	IV	1	PLCT THE CIRCUMFERENTIAL DISPLACEMENT DISTRI-BUTION.
24	55-56	12	IW	1	PLCT THE NORMAL CISPLACE- MENT DISTRIBUTION.
24	57-58	12	IPHIS	5 1	PLCT THE MERICIONAL RCTA- TICN DISTRIBUTION.
24	59-60	I 2	IPHIT	r 1	PLCT THE CIRCUMFERENTIAL ROTATION DISTRIBUTION.
24	61-62	12	IPHI	1	PLCT THE MERIDIC- CIRCUMFERENTIAL RCTATION DISTRIBUTION.

INCERT IMPERFECTION DATA HERE FOR A SATANS-II OR SATANS-IIA ANALYSIS WITH IMPERFECTIONS. INSURE FORMAT OF THE IMPERFECTION CATA IS COMPATIBLE WITH THAT SPECIFIED IN THE USER-WRITTEN SUBROUTINE IMPERF.

25 1-2 I2 IRNAGN O INCICATES THES IS THE ONLY RUN.

1 INDICATES ANCTHER RUN IS TO BE MADE. ACD ANOTHER COMPLETE SET OF CATA CARCS AFTER THIS CARD IS IRNAGN= 1.



APPENDIX D

LISTING OF NEW POLE ROUTINE FOR SATANS-IIA



CCMMCN /IBL5/IBCINL, IBCFNL

IN FCRCE
1C IF (K.NE.2.GR.(K.EQ.2.ANC.IBCINL.GE.0)) GO TC 501
DC 502 II=1,4
SL x=0.
EC 503 L=1,4
BC2 SL x=SUPX+bL(II,L,P)*GEE(L)
9C2 X(II,IKI)=SUMX
9C1 CCNINUE ں

THE FCLLCHING CARDS ARE TO BE PLACED INTO THE PMATRY SUBROUTINE

IN FRATRX
CALL EFG(2,MN)
CALL EFG(2,MN)
CALL ABINV(A,4,G1,0,DETERM,IPIVCT,INCEX,4,ISCALE)
CALL MAINN(A,4,G1,0,DETERM,IPIVCT,INCEX,4,ISCALE)
CALL MAINN(A,4,G1,DETERM,IPIVCT,INCEX,4,ISCALE)
CALL MAINN(A,4,G1,DETERM,IPIVCT,INCEX,4,ISCALE)
CALL MAINN(A,4,G1,DETERM,IPIVCT,INCEX,4,ISCALE)
CALL MAINN(A,4,G1,DETERM,IPIVCT,INCEX,4,ISCALE)
CALL MAINN(A,4,G1,DETERM,IPIVCT,INCEX,4,ISCALE)
CALL MAINN(A,4,G1,DETERM,IPIVCT,INCEX,4,ISCALE)
CALL MAINN(A,4,G1,DETERM,IPIVCT,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISCALE,INCEX,4,ISC

ں



```
L,JJ)
-TTP
}-TTC
,OETERM,IPIVOT,INDEX,4,ISCALE)
         DL (2,2,MN)=-1
L=1 4
40 F (1
1) 1) = 11
11 = 1 9 4
                                                                            [I=1,4]
                                                                                                             116=0.
116=0.
116=116.
116=116.
                                                                                               $04
903
                                                                       962
                                                                                                                                     306
```



```
CC SCB L=1,4
11F=1TP+CL1(II,L)*CL0(L,JJ)
SCB 11G=11Q+CL1(II,L)*CL2(L,JJ)
CL(II,JJ,MN)=-TTP
SC7 P(II,JJ,IJ)=TTQ
GC TO 11
```



APPENDIX E

LISTING OF CARDS FOR \overline{V} AND \overline{V} MAX

R RIGHTSON

LISTING OF CARDS FOR V AND V

S

```
STATEMENTS FOR MAIN TG CALCULATE VEAR

[C 186 N=1,MAXM]

[C 186 N=1,MAXM]

[C 186 N=1,MAXM]

[C 184 K=2,KL

[C 
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                          4.MAXM
Q.1) AVB(M)=0.
R(M)).GT.AVB(M)) AVB(M)=ABS(VBAR(M))
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                    184
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